

# Motion of bulk nanobubbles driven by thermal Marangoni flow

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Thermal Marangoni effects play important roles in bubble dynamics such as bubbles generated by water electrolysis or boiling. As macroscopic bubbles often originate from nucleated nanobubbles, it is crucial to understand how thermocapillarity operates at the nanoscale. In this study, the motion of transient bulk gas nanobubbles in water driven by a vertical temperature gradient between two solid plates is investigated using molecular dynamics simulations and analytical theory. The simulation results show that due to the thermal Marangoni force, nanobubbles move towards the hot plate at a constant velocity, similar to the behaviour of macroscale gas bubbles. However, unlike macroscale gas bubbles whose thermal conductivity and viscosity are negligible compared to those of water, the thermal conductivity and viscosity of nanoscale gas bubbles are significantly increased due to their large gas density. The thermal resistance and the slip length are also found to matter at the liquid-gas interface, though they decrease with increasing gas densities. The previous thermocapillary theory for macroscale bubbles is extended by considering all these nanoscopic effects. Expressions of the Marangoni force and the drag force are derived. By balancing the Marangoni force and the drag force, the theoretical velocity of the nanobubble migration in a thermal gradient is obtained. When using the measured transport properties of liquid, gas, and their interfaces, the theoretically obtained velocity is consistent with the result of the molecular simulations. We find that the slip length is too small to have considerable effects on nanobubble motions in the current liquid-gas system.

Key words: bubble dynamics

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# 1. Introduction

The study of transient gaseous bulk nanobubbles is of interest across various fields, such as water electrolysis, catalysis involving dissolved gases, water treatment, agriculture, medicine, and the flotation of fine particles (Zhu *et al.* 2016; Wu *et al.* 2019; Batchelor *et al.* 2022; Li & Zhang 2022; Zhang *et al.* 2024; Yadav, Nirmalkar & Ohl 2024). For instance, in the flotation process, nanobubbles offer a higher surface area to volume ratio and longer contact times for attachment to particles compared to macroscale bubbles, thereby enhancing particle–bubble collision and attachment efficiency (Li & Zhang 2022). In medicine therapy, drugs or therapeutic agents can be encapsulated within the shell of coated nanobubbles and then be released in targeted positions (Batchelor *et al.* 2022). Therefore, achieving active control over the movements of bulk nanobubbles is essential for improving the efficiency of applications involving these bubbles.

For free surface flows, the gradient of surface tension can often lead to considerable shear stress at the free surface, significantly altering flow behaviours. A concentration gradient of solutes in a solvent can cause a surface tension gradient, and the resulting flow is known as solutal Marangoni flow. For instance, a free surface can be contaminated by surface-active agents such as surfactants. When the transport of surfactants on the free surface is dominated by advection, indicated by a relatively large Marangoni number or Péclet number, the distribution of surfactants tends to be non-uniform. This nonuniformity results in a surface tension gradient and a shear stress on the free surface (Manikantan & Squires 2020). This Marangoni stress often acts against the advection flow, making surfactants crucial in stabilizing emulsions (Parker, Claesson & Attard 1994; Constante-Amores et al. 2021), damping waves on ocean surfaces (Erinin et al. 2023; Lohse 2023), translating drops and bubbles (Chen & Stebe 1997; Lohse & Zhang 2020), and deteriorating superhydrophobic surfaces (Rodriguez-Broadbent & Crowdy 2023). Notably, the effects of surfactants on the free surface, as described by the Boussinesq-Scriven model (Scriven 1960), not only alter the surface tension but also introduce a new property, i.e. surface viscosity (Zhang & Ding 2023). Apart from surfactants, ions in water can also increase or decrease the surface tension according to the empirical Hofmeister series (Boström et al. 2001). Recent studies have shown that ion-related solutal Marangoni effects significantly impact the coalescence of hydrogen bubbles and thereby the current density during water electrolysis (Park et al. 2023; Liu et al. 2023).

In non-isothermal systems, a surface tension gradient and the resulting Marangoni flow can arise from temperature variations on the free surface, as the surface tension of a liquid can change significantly with temperature. Thermal Marangoni effects are common in processes such as water electrolysis (Park et al. 2023; Bashkatov et al. 2023), drop evaporation (Ristenpart et al. 2007; Shiri et al. 2021; Kant et al. 2024), vapour bubble detachment during boiling (Christopher & Wang 2001), the stability of liquid films (Kalliadasis, Kiyashko & Demekhin 2003), the motion of drops or bubbles in liquids (Young, Goldstein & Block 1959), and plasmonic bubbles (Namura et al. 2015; Zeng et al. 2021). The surface tension of most liquids decreases with increasing temperature. However, there are so-called 'self-rewetting' fluids whose surface tension increases with increasing temperature. During boiling, where vapour bubbles are produced on the heating solid, these 'self-rewetting' fluids flow spontaneously to hotter regions on the solid due to the Marangoni force, removing dry spots on the solid and thus enhancing heat transfer (Abe 2006; Hu et al. 2014). In many non-isothermal and multicomponent liquid systems, thermal Marangoni and solutal Marangoni effects coexist. For example, during water electrolysis, Ohmic heating in the electrolyte and the electrode generates a hot spot near the electrode. This results in a thermal Marangoni force that keeps bubbles attached to the electrode (Bashkatov et al. 2023). In the meantime, solutal Marangoni forces due to ions may generate forces whose directions depend on the type of ions (Park *et al.* 2023). Another example is the periodic bouncing of plasmonic bubbles produced in a binary liquid, which is due to the competition between solutal and thermal Marangoni forces (Zeng *et al.* 2021).

The study of macroscopic bubble motion by thermal Marangoni forces was pioneered by Young et al. (1959) (referred to as YGB59 theory hereafter). Through experiments, they showed that small bubbles could be held stationary or even driven downwards against gravity by a strong negative temperature gradient. This is due to the competition between the Marangoni force and the buoyancy force. They also analytically derived the bubble velocity, under the assumption of small Reynolds numbers, small Marangoni numbers, and a spherical bubble shape. Later, this analytical solution was examined extensively by other experiments (Thompson & DeWitt 1979; Hardy 1979; Thompson, DeWitt & Labus 1980; Morick & Woermann 1993; Treuner et al. 1996). Under terrestrial conditions, the Marangoni effect is often masked by the buoyancy force exerted on the bubble due to gravity. Special interest is thus given to experiments of Marangoni-driven bubbles in micro-gravity environments (Thompson & DeWitt 1979; Thompson et al. 1980; Treuner et al. 1996) where the buoyancy force is considerably decreased. However, as the YGB59 work did not present the expressions of Marangoni forces and drag forces exerted on the bubble, approximate expressions were used in literature (Morick & Woermann 1993; Lubetkin 2003; Zeng et al. 2021). The YGB59 solution was also generalized to large Reynolds numbers (Balasubramaniam & Chai 1987) and large Marangoni numbers (Balasubramaniam & Subramanian 2000). With the development of computational methods and power, numerical simulations of the Navier-Stokes equations and the heat advection-diffusion equation were performed (Haj-Hariri, Shi & Borhan 1997; Liu & Zhang 2015; Abu-Al-Saud, Popinet & Tchelepi 2018; Meulenbroek, Vreman & Deen 2021) to study thermocapillary motions of bubbles under shape deformations in terms of large Reynolds numbers.

Bulk nanobubbles, characterized by radii smaller than a micrometre, exhibit behaviour similar to bubbles in reduced-gravity environments, as they are largely unaffected by gravity. Consequently, Marangoni forces may play a dominant role in controlling their motion. While there are extensive studies on the behaviour of macroscopic bubbles driven by temperature gradients, research on nanobubbles influenced by such gradients remains relatively sparse. Nanoscale heat transfer and flow can often behave in a manner that is different from that at the macroscale. For example, interfacial thermal resistance plays a crucial role in nanoscale heat transfer problems. Many works have shown the existence of a temperature jump across the liquid-solid surface (Barrat & Chiaruttini 2003; Alosious et al. 2019; Hadjiconstantinou & Swisher 2022), the liquid–liquid interface, and the liquid– gas interface (Patel, Garde & Keblinski 2005; Plascencia, Bird & Liang 2022; Dockar, Gibelli & Borg 2023). The slip length has also been shown to play a major role in microflows and nanoflows (Lauga, Brenner & Stone 2007; Bocquet & Charlaix 2010). Compared to the well-studied liquid-solid slip, recently there has been more interest in studying the slip length at the liquid–liquid interface (Poesio, Damone & Matar 2017; Telari, Tinti & Giacomello 2022; Hilaire et al. 2023). However, it remains unclear whether the YGB59 theory (Young et al. 1959), which assumes temperature and velocity continuity at the liquid-gas interface, works at the nanoscale.

Due to the small spatial and temporal scales of nanobubbles, and due to the unavoidable contaminants, studying clean nanobubble dynamics in experiments remains challenging (Lohse & Zhang 2015). As a result, molecular dynamics simulations have become a valuable tool for investigating nanobubbles, including nanobubble evolution in water electrolysis (Maheshwari *et al.* 2016; Zhang & Lohse 2023; Zhang *et al.* 

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2024), nanobubble cavitation (Vedadi *et al.* 2010; Shekhar *et al.* 2013; Menzl *et al.* 2016; Gao, Wu & Wang 2021), vapour nanobubble nucleation (Gallo *et al.* 2020; Tinti *et al.* 2023) and the coalescence of bulk nanobubbles (Bird, Smith & Liang 2021). Note that clean bulk nanobubbles in saturated or undersaturated environments tend to dissolve within microseconds, as predicted by the Epstein–Plesset theory (Epstein & Plesset 1950; Lohse & Zhang 2015). However, compared to the time scales of other nanobubble hydrodynamic phenomena, such as cavitation and coalescence, the time scale of dissolution is relatively longer. Therefore, on very short time scales, the effect of nanobubble dissolution, which causes a time-dependent bubble radius, may be neglected for simplicity, as demonstrated in previous studies.

To unveil the mechanism of nanobubble motions driven by thermal Marangoni flows, here the movement of a gas nanobubble between two solid plates with a temperature gradient is investigated by well-controlled molecular dynamics simulations. The previous thermal Marangoni flow theory for macroscale gas bubbles is extended by considering potentially enhancing nanoscopic interfacial effects such as the thermal resistance and the slip length across the liquid–gas interface. Expressions of the Marangoni force and the drag force are derived. By balancing the Marangoni force and the drag force, the theoretical velocity of bubble migration is determined. Finally, the simulation results are compared with the theory, finding good agreement.

This paper is organized as follows. In § 2, the procedure of molecular simulations is described. Section 3 shows the bubble trajectories from simulations. This section also includes a systematic measurement of the transport properties of water, gas, and their interfaces. Section 4 derives a model for nanobubble displacement. The comparison between simulation results and theories is then made. We conclude our paper in § 5 with a summary and an outlook.

# 2. Molecular simulations of nanobubble motion in a temperature gradient

Molecular dynamics (MD) simulations are used to simulate the motion of bulk gas nanobubbles in liquid between two solid plates whose temperatures are different. We adopt the open-source code LAMMPS (Plimpton 1995). As shown in figure 1(*a*), the molecular system consists of water molecules (represented in orange), gas atoms (represented in green), atoms of the bottom solid (represented in red) with a higher temperature  $T_h$ , and atoms of the top solid (represented in blue) with a lower temperature  $T_l$ .

The mW water potential is adopted to model water. The mW water model is a monatomic water model proposed by Molinero & Moore (2009). It uses the Stillinger–Weber potential:

$$E = \sum_{i} \sum_{j>i} \phi_2(r_{ij}) + \sum_{i} \sum_{j\neq i} \sum_{k>j} \phi_3(r_{ij}, r_{ik}, \theta_{ijk}),$$
  

$$\phi_2(r) = A\varepsilon \left[ B\left(\frac{\sigma}{r}\right)^p - \left(\frac{\sigma}{r}\right)^q \right] \exp\left(\frac{\sigma}{r-a\sigma}\right),$$
  

$$\phi_3(r, s, \theta) = \lambda\varepsilon(\cos\theta - \cos\theta_0)^2 \exp\left(\frac{\gamma\sigma}{r-a\sigma}\right) \exp\left(\frac{\gamma\sigma}{s-a\sigma}\right).$$
(2.1)

Here, A = 7.049556277, B = 0.6022245584,  $\varepsilon = 6.189$  kcal mol<sup>-1</sup>,  $\sigma = 2.3925$ Å, a = 1.80,  $\lambda = 23.15$ ,  $\gamma = 1.20$ , cos  $\theta_0 = -1/3$ , p = 4 and q = 0. Compared to other water models such as TIP4P/2005 and SPC/E, the mW water model has a good accuracy of surface tension and a low computational cost (Molinero & Moore 2009). For example,



Figure 1. (*a*) A snapshot of a gas nanobubble in water between two solid plates in MD simulations. The top plate has a lower temperature  $T_l$ , and the bottom plate has a higher temperature  $T_h$ . (*b*) Sketch of a gas nanobubble in water between two solid plates. Here,  $\gamma$  is the local surface tension of the bubble surface, and the bubble portion with a lower (higher) temperature has a higher (lower) surface tension, denoted as  $\gamma + (\gamma -)$ . This creates a Marangoni flow around the bubble, resulting in a Marangoni force  $F_{Ma}$  on the bubble.

a large time step can be chosen for the mW water model (Molinero & Moore 2009). In the current simulations, it is 8 fs.

Except for water itself, the intermolecular potentials U between *i*-type atoms and *j*-type atoms are simulated with the standard Lennard-Jones 12–6 potential:

$$U(r_{ij}) = \begin{cases} 4\varepsilon_{ij} \left[ \left( \frac{\sigma_{ij}}{r_{ij}} \right)^{12} - \left( \frac{\sigma_{ij}}{r_{ij}} \right)^6 \right] & \text{if } r_{ij} \leqslant r_{c,ij}, \\ 0 & \text{if } r_{ij} > r_{c,ij}. \end{cases}$$
(2.2)

Here,  $r_{ii}$ ,  $\varepsilon_{ii}$ ,  $\sigma_{ij}$  and  $r_{c,ij}$  are the pairwise distance, energy parameter, length parameter and cut-off distance, respectively. The complete list of parameters among water (W), gas (G) and solid (S) is given in table 1. The gas modelled by the standard 12–6 Lennard-Jones potential represents nitrogen, and it has density  $\rho_{\infty} = 1.147 \text{ kg m}^{-3}$  at 1 atm and 300 K. The molar mass of the gas is  $28 \text{ g mol}^{-1}$ . As noted earlier, clean bulk nanobubbles in saturated or undersaturated environments tend to dissolve in microseconds, as predicted by the Epstein–Plesset theory (Epstein & Plesset 1950; Lohse & Zhang 2015). We indeed find that in our molecular simulations, a nanobubble dissolves over time, leading to a timedependent bubble radius. This obviously complicates the theoretical modelling and goes beyond the scope of current work to validate the YGB59 theory (where a constant bubble radius is assumed) to the nanoscale. To circumvent this problem, one can tune the energy parameter, distance parameter and cut-off distance between gas and water to obtain a small gas solubility. In current simulations, the cut-off distance between water and gas is set to be  $2^{1/6}\sigma_{GW} = 3.4$  nm so that only the repulsive force between gas and water is effective, leading to an extremely small gas solubility. The same simulation strategy was used when simulating nanobubble cavitation (Shekhar et al. 2013).

The box has lateral size  $L_x = 19.2$  nm and  $L_y = 19.2$  nm; see figure 2(b). The height  $L_z$  of this box is fixed during the bubble movement, but it varies for different cases with

Atom type	Atom type	$\varepsilon_{ij}$ (kcal mol <sup>-1</sup> )	$\sigma_{ij}$ (angstroms)	$\gamma_{c,ij}$ (angstroms)
G	G	0.188	3.75	16.5
G	W	1.0	3.07	3.4
G	S	0.26	3.32	16.5
W	S	0.15	3.32	16.5
S	S	5.0	3.32	16.5

Table 1. Interaction parameters among water (W), gas (G) and solid (S).

Case	Bubble radius (nm)	Density $\rho (1/\text{\AA}^3)$	Box height $L_z$ (nm)	$T_l$ (K)	$T_{h}\left( \mathbf{K}\right)$	Simulation runs
1	5.0	0.0063	39.0	300	350	5
2	3.1	0.0096	38.4	300	350	9
3	5.0	0.0063	39.0	300	325	5

Table 2. Simulated three cases with different settings.



Figure 2. (*a*) Snapshots of the bubble movement in the MD simulations at three different times for case 1. (*b*) For the same simulation, the three-dimensional trace of the bubble motion is shown.

different bubble radii, as discussed below and shown in table 2. The height of each solid plate at the lower and upper ends of the domain is  $L_p = 0.96$  nm. Both the top and bottom plates are made of five layers of atoms with the same density as water for simplicity. Periodic boundary conditions are applied in the lateral directions. On the temperature conditions, the bottom hot plate is held at 350 K or 325 K, and the top cold plate is held at 300 K using the Langevin thermostat with damping factor 100 fs. A constant spring force 10 kcal mol<sup>-1</sup> Å<sup>-2</sup> is applied to each atom in the plates to tether them to their initial position. The positions and velocities of the fluid atoms (liquid and gas) are updated based on Newton's second law without extra thermostating.

The tank is initially filled with 467 200 water atoms and is isothermal with temperature 300 K. Then N water atoms next to each other centred at  $h_0 = 28.8$  nm are switched into gas atoms, which forms a nanobubble. Notably, the nanobubble is initially tethered to its initial position through a spring force so that it can only expand or shrink without

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Brownian motions. During this process, the top plate can move vertically to maintain the ambient liquid pressure at 1 atm in response to the appearance of the nanobubble. Two different situations with N = 1196 and N = 3316 are simulated. The corresponding radii of the obtained nanobubbles at equilibrium are measured to be 3.1 nm and 5.0 nm, consistent with the prediction of the real gas law (Zhang & Lohse 2023). At equilibrium, the position of the top plate barely moves, and  $L_z = 38.4$  nm for R = 3.1 nm, and  $L_z = 39$  nm for R = 5.0 nm, are measured and listed in table 2. After obtaining the initial configuration, the Langevin thermostat is applied to the two plates. The formed nanobubbles are still tethered to their initial positions for another 100 ns, during which a steady temperature gradient is produced between the two solid plates. After 100 ns, the nanobubbles are released so that they can move freely in the liquid. Due to the thermal motions of nanobubbles, for each case, several realizations (see table 2) have to be run to improve the statistics.

# 3. Results from the MD simulations

# 3.1. Nanobubble displacement

Figure 2(a) shows snapshots of the moving bubble (projected at the x-z plane) obtained from the molecular simulations at three different times for case 1. Just as for the case of macroscopic bubbles driven by a temperature difference, the nanobubble moves to the hot plate due to the thermal Marangoni force. Figure 2(b) plots the three-dimensional trajectory of the moving bubble. It can be seen that though on average the nanobubble moves downwards towards the hot plate, it experiences considerable thermal motions in all three directions during the movement. Thus, as shown in figures 3(a-c), a number of realizations have been performed to get the mean bubble trajectory, from ensemble averaging. For example, for case 1, five realizations represented by grey lines in Figure 3(a)have been conducted. The red line represents the mean value of the five realizations. The error bars stand for the standard deviations of the five samples. For case 2, the performed numbers of independent realizations are 9, as smaller bubbles are more vulnerable to thermal fluctuations, evidenced by the larger standard deviations of the samples, as shown in figure 3(b). For case 3 in figure 3(c), five realizations were performed. The obtained mean bubble distance as a function of time for the three cases is shown in figure 3(d) for comparison. The dependence of bubble distances on time is linear when the bubbles are away from the plates. This means that the Marangoni force that drives the bubble motion is balanced by the drag force. Thus thermocapillarity works down to the nanoscale. By a linear fitting of the data from 10 ns to 20 ns, the corresponding bubble velocity  $U_{MD}$ is obtained, as shown in table 3. To quantitatively compare the simulation results with the YGB59 theory (Young *et al.* 1959), one has to systematically measure the transport properties of the simulated liquid and gas.

#### 3.2. Transport properties of the simulated water and gas

Though the mW water model (Molinero & Moore 2009) adopted here is a standard model, to our knowledge, its transport properties such as surface tension and viscosity as a function of temperature are not available in the literature. A systematic measurement of these mentioned properties is thus performed. The surface tension of water is measured by the standard method using the mechanical definition of surface tension at the atomic level (see Appendix A). The value of surface tension at different temperatures ranging from 300 K to 350 K is shown in figure 4(*a*). In this temperature range, water surface tension decreases linearly with temperature, and  $d\gamma/dT = -0.082$  mN m<sup>-1</sup> K<sup>-1</sup> is obtained by

Cases	$\Delta T_{efj}$ (K)	$(WK^{-1}m^{-1})$	$\mu^i$ (mPas)	b <sup>o</sup> (nm)	b <sup>i</sup> (nm)	$G (10^{-8} \text{m}^2 \text{KW}^{-1})$	$U_{MD}$ (ms <sup>-1</sup> )	$U_{Macro}$ (ms <sup>-1</sup> )	$U_{Nano-NS} \ ({ m ms}^{-1})$	$U_{Nano-FS}$ $(ms^{-1})$
1	41	0.0466	0.0270	0.3	0.61	2.0	0.62	0.91	0.67	0.68
2	41	0.054	0.0326	0.15	0.52	1.1	0.43	0.56	0.40	0.42
3	21	0.0466	0.0270	0.3	0.61	2.0	0.38	0.45	0.34	0.35

Table 3. Comparisons among the bubble velocity results of MD simulations  $(U_{MD})$ , the theoretical velocity ignoring the gas thermal conductivity and the viscosity  $(U_{Macro})$ , and the revisited theoretical velocity  $U_{Nano-NS}$  ( $U_{Nano-FS}$ ) considering nanoscale effects and without (with) the slip length. Here,  $\Delta T_{eff}$  is the effective temperature difference after considering the solid–liquid thermal resistance.



Figure 3. Trajectories of the moving bubbles in the molecular simulations for the three different cases. Grey lines represent different realizations in each case. The solid coloured lines are the mean trajectories of all the realizations, with the error bars representing the standard deviations: (*a*) case 1, bubble radius R = 5.0 nm and  $\Delta T = 50$  K; (*b*) case 2, bubble radius R = 3.1 nm and  $\Delta T = 50$  K; (*c*) case 3, bubble radius R = 5.0 nm and  $\Delta T = 25$  K; (*d*) mean bubble trajectory for the mentioned three cases, for comparisons.

a linear fitting as shown in figure 4(*a*). The shear viscosity of water is measured by the Green–Kubo method as also explained in Appendix A. We obtain that the dynamical viscosity  $\mu^o$  (the superscript <sup>o</sup> is for the liquid, and the superscript <sup>i</sup> is for the inner fluid) of the mW water model at 300 K is approximately 0.35 mPa s, which underpredicts the experimental value compared to other water models such as TIP4P/2005 and SPC/E

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Figure 4. (*a*) Surface tension of the mW water model as a function of temperature. Symbols are MD results. The dashed lines show the linear fitting. (*b*) Viscosity (left-hand axis) and thermal conductivity (right-hand axis) for the mW water model. (*c*) Viscosity (left-hand axis) and thermal conductivity (right-hand axis) of gas as a function of gas density. (*d*) Liquid–gas slip (left-hand axis) and thermal resistance (right-hand axis) as a function of gas density.

(González & Abascal 2010). The dynamic viscosity of water also exhibits a temperature dependence like surface tension. However, the dependence is weak in the temperature range 300–350 K. Therefore we take the dynamic viscosity of water to be the mean dynamic viscosity  $\mu^o = 0.27$  mPa s in this temperature range, for simplicity. In principle, the thermal conductivity of water can also be measured by the Green–Kubo method, but this method is known to converge poorly for thermal conductivity (Sirk, Moore & Brown 2013). Thus here the thermal conductivity of water  $k^o$  is measured by the non-equilibrium heat flux method (Wirnsberger, Frenkel & Dellago 2015), as shown in Appendix A. Within the simulated temperature range (300–500 K), the thermal conductivity of water hardly varies, thus the mean value  $k^o = 0.34$  W m<sup>-1</sup> K<sup>-1</sup> is used. The mW water model underpredicts the experimental value of water thermal conductivity, while other water models TIP4P/2005 and SPC/E overpredict the experimental value (Sirk *et al.* 2013).

Normally for macroscopic gas bubbles, the heat conduction and the flow inside the bubble are ignored because the heat conductivity and the viscosity of the gas are both small compared to those for water outside. For a dilute gas, based on the kinetic theory, its thermal conductivity and viscosity depend only weakly on temperature, and are independent of gas pressure. Under these conditions, one may ignore the gas and simply take the property of water to calculate the bubble velocity. Based on the YGB59 theory (Young *et al.* 1959), i.e. (4.28), the theoretical bubble velocities for the three cases are





Figure 5. (a) Snapshot of the liquid–gas system in MD simulations to measure the liquid–gas thermal resistance. (b) The temperature profile for gas density  $0.0036 \text{ Å}^3$ . (c) The temperature profile for gas density  $0.0068 \text{ Å}^3$ . (d) The temperature profile for gas density  $0.0096 \text{ Å}^3$ . The symbols are MD results, and the red lines are linear fittings.

then calculated to be 0.91 m s<sup>-1</sup>, 0.56 m s<sup>-1</sup> and 0.45 m s<sup>-1</sup> (see  $U_{Macro}$  in table 3), which, however, considerably overpredict the simulated bubble velocity  $U_{MD}$ .

There are multiple factors that may cause the mismatch between the MD results and the YGB59 theory. For nanobubbles whose pressure can be as high as dozens of MPa, the predictions of the gas viscosity and thermal conductivity, based on the kinetic theory, may be wrong. Indeed, we find that the thermal conductivity and viscosity of gas strongly depend on the gas density as shown in figure 4(c), measured by the method discussed below. Another reason may be that the temperature and velocity continuity condition at the bubble surface breaks down.

## 3.3. Thermal resistance and slip length across the liquid–gas interface

Many researchers have shown the existence of a temperature jump across a liquid–solid interface (Barrat & Chiaruttini 2003; Alosious *et al.* 2019; Hadjiconstantinou & Swisher 2022), a liquid–liquid interface (Patel *et al.* 2005) and a liquid–gas interface (Patel *et al.* 2005; Plascencia *et al.* 2022; Dockar *et al.* 2023). A velocity jump may also exist at a liquid–liquid interface (Poesio *et al.* 2017; Telari *et al.* 2022; Hilaire *et al.* 2023).

To check whether there is thermal resistance between the water–gas interface and the water–solid interface, we use the setting as shown in figure 5(a), where both gas (in green) and water (in dark yellow) are confined between two plates held at different temperatures.



Figure 6. (a) Snapshot of the Couette flow to measure the gas–liquid slip. (b) The velocity profile for gas density  $0.0036 \mathring{A}^3$ . (c) The velocity profile for gas density  $0.0063 \mathring{A}^3$ . (d) The velocity profile for gas density  $0.0096 \mathring{A}^3$ . The symbols are MD results, and the red lines are linear fittings.

While the number of water atoms is fixed in the box, the number of gas atoms is varied to have different gas densities. After equilibrium, the temperature profile is obtained. For a low gas number density  $\rho = 0.0036 \text{ Å}^3$ , as shown in figure 5(*b*), we observe noticeable temperature jumps at the liquid–solid, liquid–gas and gas–solid interfaces. As the gas–solid interface is not relevant for the Marangoni-driven bubble motion, only the thermal resistance for the liquid–solid interface and the liquid–gas interface are calculated. To obtain the temperature jump, we interpolate the linear fitting of the temperature profile into the interfaces: a temperature jump 3.4 K is obtained for the liquid–solid interface (at z = 6.2 nm). By its definition (4.7), the thermal resistance of the liquid–solid interface is calculated to be  $1.30 \times 10^{-8} \text{ m}^2 \text{ K W}^{-1}$ , and for the liquid–gas interface, it is  $4.16 \times 10^{-8} \text{ m}^2 \text{ K} W^{-1}$ . The order of obtained thermal resistance is consistent with results reported in the liquid and in the gas, the thermal conductivity of the gas of density  $\rho = 0.0036 \text{ Å}^3$  is  $0.028 \text{ W m}^{-1} \text{ K}^{-1}$ .

For the gas density the same as case 1 ( $\rho = 0.0063 \text{\AA}^3$ ) and case 2 ( $\rho = 0.0096 \text{\AA}^3$ ), the temperature profiles in figures 5(c) and 5(d) show smaller temperature jumps

7.7 K and 5.3 K at the liquid–gas interface. The thermal resistances of the liquid– gas are calculated to be  $2.0 \times 10^{-8}$  m<sup>2</sup> K W<sup>-1</sup> and  $1.1 \times 10^{-8}$  m<sup>2</sup> K W<sup>-1</sup>. The thermal resistance is thus reduced by increasing the gas density, which is similar to the theoretical prediction of the solid–gas thermal resistance (Chen *et al.* 2022) and molecular simulations of liquid–vapour thermal resistance (Holyst & Litniewski 2008). The thermal conductivity of the gas with a higher density is, however, increased. For  $\rho = 0.0063 \text{Å}^3$ , it is  $0.0466 \text{ W m}^{-1} \text{ K}^{-1}$ , and for  $\rho = 0.0096 \text{Å}^3$ , it is  $0.0593 \text{ W m}^{-1} \text{ K}^{-1}$ . The relation between gas thermal conductivity or liquid–gas thermal resistance and gas density is shown in figures 4(c) and 4(d). One can also use the heat flux method (Wirnsberger *et al.* 2015) to measure the gas thermal conductivity as a function of gas density, and we have checked that both methods produce the same results. We also note that the temperature jumps across the liquid–solid interface indeed increase with an increasing heat flux, as shown by figures 5(b-d), and the mean liquid–solid thermal resistance of the three cases is  $1.22 \times 10^{-8} \text{ m}^2$  K W<sup>-1</sup>, which is used later to estimate the effective temperature difference that drives nanobubble motions (see  $T_{eff}$  in table 3).

In terms of the slip length between liquid and gas, we use a similar configuration, as shown in figure 6(*a*). Here, the left-hand plate can move in the *x* direction with velocity  $5 \times 10^{-4}$  m s<sup>-1</sup>, while the right-hand plate is fixed. The velocity profiles of this Couette flow are shown in figures 6(*b*-*d*) for three different gas densities. One can see that there are indeed velocity jumps at the liquid–gas interface. The velocity jump decreases with increasing gas density. By its definition (B1) shown in Appendix B, the slip length for the gas (liquid) phase  $b^i$  ( $b^o$ ) is measured to be 1.1 nm (1.3 nm), 0.61 nm (0.3 nm), and 0.52 nm (0.15 nm), respectively for the three different gas densities. So the slip lengths  $b^i$  and  $b^o$ decrease with increasing gas densities, as shown in figure 4(*d*). The gas viscosity can be obtained by the ratio of the slopes of the red fitting lines based on the continuity of the shear stress. The obtained gas viscosities are 0.0229 mPa s, 0.0270 mPa s and 0.0326 mPa s, respectively, for the three different gas densities, thus the gas viscosities increase with increasing gas densities. This relation is also plotted in figure 4(*c*).

After all, we discovered that the thermal conductivity and viscosity of gas nanobubbles have been enhanced a lot by the large gas density on the nanoscale. It was also found that temperature jumps and velocity jumps coexist at the bubble surface, which invalidates the assumption of temperature and velocity continuities at the bubble surface adopted by the classic YGB59 theory (Young *et al.* 1959). Thus the YGB59 theory has to be generalized. Notably, though we have obtained thermal resistance and slip length from our simulations whose values are of the same order as reported values in the literature, we have used an ideal potential to describe the water–gas intermolecular interactions compared to other more realistic systems (Plascencia *et al.* 2022; Dockar *et al.* 2023; Hilaire *et al.* 2023). However, the existence of thermal resistance and slip at the liquid–gas interface is generally true.

#### 4. Extension of the YGB59 theory towards the nanoscale

Though Young *et al.* (1959) (YGB59 theory) derived the correct expression of bubble velocity, their work did not include the expression of Marangoni forces and drag forces. This has led to some approximate expressions in the literature (Morick & Woermann 1993; Lubetkin 2003; Zeng *et al.* 2021), which are valid only at the macroscale. Here, the YGB59 equation is revisited, and more general expressions for Marangoni forces and drag forces are obtained, which show important nanoscale effects, such as those of the thermal resistance and the slip at the interface.

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The governing dimensionless numbers for both fluids (liquid and gas) are the Reynolds number  $Re = \rho u_0 R/\mu$  and the Marangoni number  $Ma = (d\gamma/dT)(dT/dz)(R^2/(\mu\alpha))$ . Here,  $\rho$ ,  $\mu$ ,  $u_0$ , R,  $\gamma$  and  $\alpha$  are the density, dynamic viscosity, velocity, radius of the nanobubble, surface tension and thermal diffusivity, respectively. Assuming that Re and Ma are much smaller than unity for both fluids, which is valid at the small scale (small R) of nanobubbles, the momentum equation and continuity equation for both fluids are

$$\mu \, \boldsymbol{\nabla}^2 \boldsymbol{u} = \nabla \boldsymbol{p}, \tag{4.1}$$

$$\boldsymbol{\nabla} \cdot \boldsymbol{u} = 0, \tag{4.2}$$

respectively. In the momentum equation, the unsteady term  $\rho \partial u/\partial t$  is dropped. This is justified because the ratio of the time scale required for the water velocity to adjust to the changes in the bubble position ( $\rho R^2/\mu$ ) to the time scale of bubble movement ( $R/u_0$ ), i.e. Re, is small. In fact, the largest Reynolds number in our simulations is evaluated to be 0.01 for case 1. Therefore, the water velocity field may be assumed to be quasi-static. The heat transfer equation for both fluids is

$$\nabla^2 T = 0. \tag{4.3}$$

The general solution to the temperature field for both fluids in spherical coordinates is

$$T = A_1 + \frac{A_2}{r} + A_3 r \cos \theta + \frac{A_4}{r^2} \cos \theta.$$
 (4.4)

Here,  $A_i$  (i = 1, 2, 3, 4) is the coefficient to be determined by the boundary conditions. At the far field, the temperature distribution  $T^o$  for the outer fluid (i.e. water here – the superscript <sup>o</sup> is for the liquid, and the superscript <sup>i</sup> is for the inner fluid, i.e. gas hereafter) is

$$T^o = T_0 + \Gamma r \cos \theta, \tag{4.5}$$

where  $T_0$  is the temperature at the centre of the bubble, and  $\Gamma$  is the temperature gradient. At the fluid–fluid interface, the continuity of the heat flux requires

$$k^{o}\frac{\partial T^{o}}{\partial r} = k^{i}\frac{\partial T^{i}}{\partial r},\tag{4.6}$$

where we have assumed a spherical bubble with the coordinate system at its centre. For the temperature condition across the interface, motivated by the observation from our simulations, we adopt the temperature jump condition across the interface instead of the continuity of temperature, so that one can have

$$T^{o} - T^{i} = k^{o} \frac{\partial T^{o}}{\partial r} G, \qquad (4.7)$$

where G is the thermal resistance.

Using the above temperature boundary conditions, the temperature distribution can be obtained for both fluids:

$$T^{o} = T_{0} + \Gamma r \cos \theta + \frac{(k^{o} - k^{i})R + k^{o}k^{i}G}{(2k^{o} + k^{i})R + 2k^{o}k^{i}G} \frac{\Gamma R^{3} \cos \theta}{r^{2}}, \quad r \ge R,$$
  

$$T^{i} = T_{0} + \frac{3k^{o}R}{(2k^{o} + k^{i})R + 2k^{o}k^{i}G} \Gamma r \cos \theta, \quad r < R.$$
(4.8)

In terms of the velocity field, the usual way to solve (4.1) and (4.2) is by introducing the Stokes stream function  $\psi(r, \theta)$  (Bird, Stewai & Lightfoot 2006)

$$u_r = -\frac{1}{r^2 \sin \theta} \frac{\partial \psi}{\partial \theta},$$
  

$$u_\theta = \frac{1}{r \sin \theta} \frac{\partial \psi}{\partial r}.$$
(4.9)

Equation (4.1) is then rewritten in terms of the Stokes stream function  $\psi$  as

$$E^4\psi = 0,$$
 (4.10)

where  $E^2 = (\partial/\partial r^2) + (\sin \theta/r^2)(\partial/\partial \theta)((1/\sin \theta)(\partial/\partial \theta))$ . The general solution to (4.10) for both fluids is

$$\psi^{o} = \left(C_{1}r^{-1} + C_{2}r + C_{3}r^{2} + C_{4}r^{4}\right)\sin^{2}\theta, \quad r \ge R,$$
(4.11)

$$\psi^{i} = \left( D_{1}r^{-1} + D_{2}r + D_{3}r^{2} + D_{4}r^{4} \right) \sin^{2}\theta, \quad r < R.$$
(4.12)

The resulting expressions for the velocities are

$$u_r^o = -2\cos\theta \left(\frac{C_1}{r^3} + \frac{C_2}{r} + C_3 + C_4 r^2\right),$$
(4.13)

$$u_{\theta}^{o} = \sin \theta \left( -\frac{C_{1}}{r^{3}} + \frac{C_{2}}{r} + 2C_{3} + 4C_{4}r^{2} \right), \qquad (4.14)$$

$$u_r^i = -2\cos\theta \left(\frac{D_1}{r^3} + \frac{D_2}{r} + D_3 + D_4 r^2\right),$$
(4.15)

$$u_{\theta}^{i} = \sin\theta \left( -\frac{D_{1}}{r^{3}} + \frac{D_{2}}{r} + 2D_{3} + 4D_{4}r^{2} \right).$$
(4.16)

These factors *C* and *D* are determined by the velocity boundary conditions. We assume that the bubble has velocity  $U \equiv -u_{\infty}$  ( $u_{\infty} > 0$ ). Taking the reference frame with the bubble, the far-field liquid velocity will be  $u_r = u_{\infty} \cos \theta$  and  $u_{\theta} = -u_{\infty} \sin \theta$ . Thus the far-field stream function will be

$$\psi^o = -\frac{1}{2}u_{\infty}r^2\sin^2\theta, \quad r \to \infty, \tag{4.17}$$

so  $C_4 = 0$  and  $C_3 = -(1/2)u_{\infty}$  using (4.11). For the flow inside the nanobubble, the velocity at r = 0 has to be finite, so that  $D_1 = 0$  and  $D_2 = 0$  using (4.12). Furthermore, there are no masses across the bubble surface, and no bubble shape changes, so that  $u_r^o = 0$  and  $u_r^i = 0$  at r = R. This leads to  $C_1 = (u_{\infty}R - 2C_2)R^2/2$  and  $D_3 = -R^2D_4$ .

For boundary conditions at the liquid–gas interface, the shear stress continuity is obeyed (Leal 2007):

$$\tau^{o}_{r\theta} - \tau^{i}_{r\theta} = -\nabla_{s}\gamma. \tag{4.18}$$

Here, one can obtain  $-\nabla_s \gamma = -(1/R)(\partial \gamma/\partial \theta) = -(1/R)(\partial \gamma/\partial T)(\partial T/\partial \theta) = (\partial \gamma/\partial T)(3k^o R + 3k^o k^i G)/((2k^o + k^i)R + 2k^o)k^i G\Gamma \sin \theta$ , knowing the temperature distribution by (4.8). Later, we define  $A = (\partial \gamma/\partial T)(3k^o R + 3k^o k^i G)/((2k^o + k^i)R + 2k^o k^i G\Gamma)$  such that we can simply write  $-\nabla_s \gamma = A \sin \theta$ .

A number of recent works have shown that there exists slip at a fluid–fluid interface (Poesio *et al.* 2017; Telari *et al.* 2022; Hilaire *et al.* 2023). As seen from the simulation of the two-phase Couette flow in figure 6, this is indeed the case for the liquid–gas interface

in our system. Thus it is natural to include slip in the theoretical model for the bubble velocity. However, how to model the fluid–fluid slip condition is not a well-explored topic. Even for the liquid–solid slip, the frequently used Navier's slip condition (Zhang 2024), which assumes that the slip velocity is proportional to shear stress at the wall, is empirical (Leal 2007). Thus at first, we still use the classic no-slip condition to derive the theoretical bubble velocity. Later, we then try to use empirical fluid–fluid slip models to see any improvements in the predictions.

The no-slip condition at the fluid-fluid interface is

$$u^o_\theta = u^i_\theta. \tag{4.19}$$

Using the shear stress condition (4.18) and the no-slip condition (4.19), one can obtain

$$D_4 = \frac{4C_2 - 3u_\infty R}{4R^3},\tag{4.20}$$

$$C_2 = \frac{6\mu^o u_\infty R^2 + 9\mu^i u_\infty R^2 - 2AR^3}{12\mu^o R + 12\mu^i R}.$$
(4.21)

Now the Marangoni force exerted on the bubble can be evaluated by integrating the shear stress:

$$F_{Ma} = -2\pi R^2 \int_0^{\pi} \tau_{r\theta} \sin^2 \theta \, d\theta = -8\pi \, \mu^o \left( u_\infty R - 2C_2 \right)$$
$$= -\frac{4\pi}{3} \mu^o \frac{2AR^2 - 3\mu^i u_\infty R}{\mu^o + \mu^i}.$$
(4.22)

Similarly, by integrating the normal stress  $\sigma_{rr} = -p + 2\mu(\partial u_r/\partial r)$ , the pressure drag force experienced by the bubble is

$$F_{P} = 2\pi R^{2} \int_{0}^{\pi} \sigma_{rr} \sin \theta \cos \theta \, d\theta = 8\pi \mu^{o} \left( u_{\infty} R - C_{2} \right)$$
$$= 2\pi \mu^{o} \frac{2AR^{2} + 6\mu^{o} u_{\infty} R + 3\mu^{i} u_{\infty} R}{3\mu^{o} + 3\mu^{i}}.$$
(4.23)

One can see that the eventual expression of the pressure drag depends on  $C_2$ , which again depends on the boundary condition at the liquid–gas interface. For a clean bubble with a negligible gas viscosity,  $\tau_{r\theta} = 0$  so that  $C_2 = u_{\infty}R/2$  and  $F_P = 4\pi \mu^o u_{\infty}R$ , which recovers the classic drag on a conventional bubble. For our case,  $C_2$  is different. By letting  $F_{Ma} + F_P = 0$ , one finds that  $C_2 = 0$ . Using (C2), the velocity of the bubble  $(U \equiv -u_{\infty})$  is obtained as

$$U = -\frac{2R}{2\mu^o + 3\mu^i} \frac{k^o R + k^o k^i G}{\left(2k^o + k^i\right) R + 2k^o k^i G} \frac{\partial \gamma}{\partial T} \Gamma.$$
 (4.24)

If there is no temperature jump (G = 0), then (4.24) reduces to the result of YGB59 theory (Young *et al.* 1959):

$$U = -\frac{2R}{2\mu^o + 3\mu^i} \frac{k^o}{2k^o + k^i} \frac{\partial\gamma}{\partial T} \Gamma.$$
(4.25)

In this case, the Marangoni force and drag force are

$$F_{Ma} = -4\pi \mu^o \frac{R^2 \frac{\partial \gamma}{\partial T} \frac{2k^o}{2k^o + k^i} \Gamma - \mu^i u_\infty R}{\mu^o + \mu^i}, \qquad (4.26)$$

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$$F_P = 2\pi\mu^o \frac{R^2 \frac{\partial \gamma}{\partial T} \frac{2k^o}{2k^o + k^i} \Gamma + 2\mu^o u_\infty R + \mu^i u_\infty R}{\mu^o + \mu^i}.$$
(4.27)

For macroscale gas bubbles whose viscosity and thermal conductivity are usually ignored, the velocity becomes

$$U = -\frac{R}{2\mu^o} \frac{\partial \gamma}{\partial T} \Gamma, \qquad (4.28)$$

and the forces are  $F_{Ma} = -4\pi (\partial \gamma / \partial T) \Gamma R^2$  and  $F_P = 4\pi \mu^o u_\infty R + 2\pi (\partial \gamma / \partial T) \Gamma R^2$ . Thus the drag force is increased by the presence of the surface tension gradient. This extra term  $2\pi (\partial \gamma / \partial T) \Gamma R^2$  was not included in previous works about bubble motions in temperature gradients on the macroscale (Morick & Woermann 1993; Lubetkin 2003; Zeng *et al.* 2021). Reassuringly, if one investigates the total force  $F_{Ma} + F_P$  on the bubble, then it is  $4\pi \mu^o u_\infty R - 2\pi (\partial \gamma / \partial T) \Gamma R^2$ , which is the same for the previous study and the current study.

With all the measured transport properties of liquid, gas, and their interfaces, as summarized in table 3, now the theoretical velocity of nanobubbles without slip  $U_{Nano-NS}$  (see (4.24)) can be evaluated. As shown in table 3,  $U_{Nano-NS}$  values for the three cases agree reasonably well with the simulation results  $U_{MD}$ . This highlights the consideration of the enhanced nanoscopic effects such as gas thermal conductivity, gas viscosity and liquid–gas thermal resistance for accurate descriptions of the nanobubble motion in a temperature gradient.

However, will the consideration of the slip improve the prediction even more? To answer this question, we have to use 'empirical' slip conditions for the fluid–fluid interface to include slip in the theoretical model of bubble velocity. In analogy to Navier's slip condition for a liquid–solid interface, the slip condition for a fluid–fluid interface is proposed to be (Hilaire *et al.* 2023)

$$u_1 - u_2 = \frac{b_1}{\mu_1} \tau_1 = \frac{b_2}{\mu_2} \tau_2, \tag{4.29}$$

where  $u_1$ ,  $b_1$  and  $\tau_1$  ( $u_2$ ,  $b_2$  and  $\tau_2$ ) are the interfacial velocity, slip length and interfacial shear stress of fluid 1 (fluid 2). For a clean surface where  $\tau_1 = \tau_2$ , (4.29) means  $b_1/b_2 = \mu_1/\mu_2$ . However, in our case,  $\tau_1 \neq \tau_2$  due to the extra Marangoni stress at the interface. Thus to account for this inconsistency, an extended slip model for the fluid–fluid interface may be (see Appendix B)

$$u_{\theta}{}^{o} - u_{\theta}{}^{i} = \frac{b^{o}}{\mu^{o}} \tau^{o}_{r\theta} + \frac{b^{i}}{\mu^{i}} \tau^{i}_{r\theta}.$$
(4.30)

Note that the definition of slip length in (4.30) is different from (4.29).

Using this boundary condition, the obtained bubble velocity with finite slip  $U_{Nano-FS}$  is (see Appendix C)

$$U = -\frac{2R}{2\mu^{0} + 3\mu^{i}\frac{1+2b^{o}/R}{1+3b^{i}/R}} \frac{k^{o}R + k^{o}k^{i}G}{(2k^{o} + k^{i})R + 2k^{o}k^{i}G} \frac{\partial\gamma}{\partial T} \Gamma.$$
 (4.31)

One can see that the slip from the inner phase  $b^i$  may speed up the bubble movement since the inner circulation is reduced by the slip. In contrast, the slip from the outer phase may slow down the bubble since the also reduced outer flow is the reason for bubble movement.

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However, using the slip lengths  $b^i$  and  $b^o$  measured from independent MD simulations (see table 3), the consideration of slip does not change the theoretical prediction too much in our case (see  $U_{Nano-FS}$  in table 3) because b/R is small. But a large liquid–liquid slip may also exist in other systems, such as at the interface between different types of polymer melts (Himmel & Wagner 2013), so the consideration of slip may then become necessary.

Notably, the small difference between  $U_{Nano}$  and  $U_{MD}$  may be attributed to several reasons. First, the water viscosity shows a weak temperature dependence in the temperature range from 300 K to 350 K. However, we have used the mean viscosity in this temperature range for simplicity. Second, the Brownian motions of bubbles are strong in MD simulations, and only a limited number of realizations have been performed due to the high computational costs. Third, the solid plates may affect the flow field, though we have used the data of bubble displacements away from the plates.

# 5. Conclusions and outlooks

In summary, motivated by the fact that macroscopic bubbles often originate from nucleated nanobubbles, the nanobubble motion in a temperature gradient is investigated by molecular dynamics simulations and approximate analytical calculations. Unlike macroscopic bubbles, whose thermal conductivity and viscosity are normally ignored compared to those of water, the simulation results show that the gas thermal conductivity and viscosity of nanobubbles are enhanced to a great extent due to the high internal pressure and the high gas density, and have to be considered. The thermal resistance and slip length across the interface are also found to exist on the liquid–gas interface. The thermocapillary theory by Young et al. (1959) for macroscale bubbles is thus extended by considering those effects. For the liquid–gas slip condition, an empirical model is used. Expressions for the Marangoni force, the drag force and the bubble migration velocity are derived. The theoretical bubble velocity evaluated using the measured transport properties of liquid, gas, and their interfaces matches well with the results of molecular simulations. This work highlights the need to consider nanoscopic effects for accurate predictions of nanobubble motions driven by thermal Marangoni flows. However, the small slip length in the current liquid–gas system does not have considerable effects on nanobubble motions.

Our findings have important implications for controlling nanobubble motions during water electrolysis and boiling where current models have not taken the nanoscopic effects into consideration. For example, for the nanobubble detachment, one has to estimate the thermal Marangoni force that keeps nanobubbles attached to the solid (Park *et al.* 2023; Bashkatov *et al.* 2023). Our results above have proven that nanoscopic effects may change the thermal Marangoni force significantly at the nanoscale. Another example is the shear flow over the substrate, which is often used to detach bubbles (Darband, Aliofkhazraei & Shanmugam 2019). The enhanced gas viscosity at the nanoscale will certainly change the drag force on surface nanobubbles. Also, for the coalescence of nanobubbles, the enhanced gas viscosity may have some effects on the film drainage between two nanobubbles and then the bridge growth dynamics (Eggers, Sprittles & Snoeijer 2024). In the future, it is also important to see the dynamics of growing or dissolving nanobubbles under a temperature gradient. Finally, the current slip model is empirical. More research is required to clarify the liquid–liquid slip condition in the future as well.

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#### Appendix A: Measurements of transport properties

The surface tension  $\gamma$  is obtained by its mechanic definition. For a planar liquid–vapour interface perpendicular to the z axis of the molecular simulation,  $\gamma$  is given by

$$\gamma = \int_{-\infty}^{\infty} \left( S_N - S_T \right) \, \mathrm{d}z. \tag{A1}$$

Here, S is the atomic definition of the pressure tensor in molecular simulations,  $S_N$  is the normal component of S, and  $S_T$  is the tangential component. The dynamic viscosity is obtained from the Green–Kubo expression

$$\mu = \frac{V}{k_B T} \int_0^\infty \left\langle S_{zx}(t) \; S_{zx}(0) \right\rangle \, \mathrm{d}t,\tag{A2}$$

where V is the volume of the liquid, and  $S_{zx}$  is the off-diagonal stress. To calculate the viscosity numerically in MD simulations, a cubic box of molecules undergoing thermal motions is simulated with periodic conditions in all three directions. The temporal correlation of the shear stress, which decays to zero with time, is then obtained from the MD simulation (see figure 7*a*) and integrated to obtain the viscosity (see figure 7*b*).

The heat flux method is used to calculate the thermal conductivity of water, as described by Wirnsberger *et al.* (2015). A periodic cubic box with length 4.8 nm filled with 3703 water atoms is simulated. An energy current  $F = 1.38 \times 10^{-7}$  J s<sup>-1</sup> is subtracted and added into the specific region of the system to sustain a temperature gradient as shown in figure 7(*c*). The thermal conductivity can then be calculated as k = F/(2A dT/dz), where *A* is surface area, and the factor 1/2 is due to the periodicity condition (Wirnsberger *et al.* 2015).



Figure 7. (*a*) Temporal correlations of the stress when measuring the mW water model with temperature 275.6 K. (*b*) Integrating the temporal correlation. (*c*) Temperature distribution in the heat flux method.

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Figure 8. Two-phase Couette flow between two plates: (a) no-slip, (b) finite slip.

# Appendix B: Fluid-fluid slip condition

For a clean surface, the stress continuity of  $\tau_1 = \tau_2$  means  $b_1/b_2 = \mu_1/\mu_2$  in (4.29). However, in this way, the defined  $b_1$  and  $b_2$  may not be the intrinsic slip of the fluid– fluid interface, since  $\tau_1 \neq \tau_2$  if there is other shear stress such as the Marangoni stress at the interface. Here, an extension of (4.29) is proposed. Unlike (4.29), where the definition of slip velocity is the same for both fluids (i.e.  $u_1 - u_2$ ), it may be different for two fluids, in fact. For example, consider a simple two-phase Couette flow shown in figure 8. The interfacial velocity is  $u_1 = u_2 = u_0$  when there is no slip at the fluid–fluid interface, as shown in figure 8(a). When there is a slip, as shown in figure 8(b),  $u_1 > u_0$  and  $u_2 < u_0$ . So the slip velocity for fluid 1 is  $u_1 - u_0$ , while it is  $u_2 - u_0$  for fluid 2. Thus the slip condition may be

$$u_1 - u_0 = \frac{b_1}{\mu_1} \tau_1,$$
  

$$u_2 - u_0 = -\frac{b_2}{\mu_2} \tau_2.$$
 (B1)

The difference between  $u_1$  and  $u_2$  is thus

$$u_1 - u_2 = \frac{b_1}{\mu_1} \tau_1 + \frac{b_2}{\mu_2} \tau_2.$$
 (B2)

For a clean surface,  $\tau_1 = \tau_2$ , (B2) becomes

$$u_1 - u_2 = \left(\frac{b_1}{\mu_1} + \frac{b_2}{\mu_2}\right)\tau_1 = \left(\frac{b_1}{\mu_1} + \frac{b_2}{\mu_2}\right)\tau_2,$$
 (B3)

which is effectively the same as (4.29). Note that the value of the slip length depends on how exactly it is defined. The factor in front of the stress will be the same for (4.29) and (B2) when  $\tau_1 = \tau_2$ . The extended model may be general as there is no requirement of  $\tau_1 = \tau_2$  and  $b_1/\mu_1 = b_2/\mu_2$ .

To extract the slip length from our independent MD simulations shown in figure 6(*a*), we need to know  $u_0$ . It is simply given by  $u_0 = U/(1 + \mu_2 h_1/(\mu_1 h_2))$ , where  $h_1 = 6.2$  nm and  $h_2 = 3.4$  nm are the heights of the two fluid domains. For the case of a gas density  $0.0096 \text{\AA}^3$  shown in figure 6(*d*),  $\mu_2/\mu_1 = 0.12$  so that  $u_0 = 4.1 \times 10^{-4}$  m s<sup>-1</sup>. By the definition of slip,  $b^o = 0.15$  nm and  $b^i = 0.52$  nm are obtained.

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### Appendix C: Bubble velocity with finite slip

Using the shear stress condition (4.18) and the finite slip condition (4.30), one can obtain

$$D_4 = \frac{C_2 \left(\frac{2}{R} + \frac{6b^o}{R^2}\right) - \left(\frac{3}{2} + \frac{3b^o}{R}\right)u_{\infty}}{6b^i R + 2R^2},$$
(C1)

$$C_{2} = \frac{6\mu^{0}u_{\infty}\left(3b^{i}R + R^{2}\right) + 3\mu^{i}u_{\infty}\left(3R^{2} + 6b^{o}R\right) - 2A\left(R + 3b^{i}\right)R^{2}}{12\mu^{0}\left(R + 3b^{i}\right) + 12\mu^{i}\left(R + 3b^{o}\right)}.$$
 (C2)

Now the Marangoni force exerted on the bubble can be evaluated by integrating the shear stress:

$$F_{Ma} = -2\pi R^2 \int_0^{\pi} \tau_{r\theta} \sin^2 \theta \, d\theta = -8\pi \mu^o \left( u_\infty R - 2C_2 \right)$$
$$= -\frac{4\pi}{3} \mu^o \frac{2A \left( R + 3b^i \right) R^2 - 3\mu^i u_\infty R^2}{\mu^0 \left( R + 3b^i \right) + \mu^i \left( R + 3b^o \right)}.$$
(C3)

The pressure drag force experienced by the bubble is

$$F_P = 2\pi R^2 \int_0^{\pi} \sigma_{rr} \sin \theta \cos \theta \, d\theta = 8\pi \mu^o \left( u_{\infty} R - C_2 \right)$$
  
=  $2\pi \mu^o \frac{2A \left( R + 3b^i \right) R^2 + 6\mu^o u_{\infty} \left( R^2 + 3b^i R \right) + 3\mu^i u_{\infty} \left( R^2 + 6b^o R \right)}{3\mu^0 \left( R + 3b^i \right) + 3\mu^i \left( R + 3b^o \right)}.$  (C4)

The condition  $F_{Ma} + F_P = 0$  leads to  $C_2 = 0$  and thus the expression of bubble velocity with finite slip (4.31).

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